

(19)



Europäisches Patentamt
European Patent Office
Office européen des brevets



(11) Publication number:

0 313 644 B1

(12)

EUROPEAN PATENT SPECIFICATION

(45) Date of publication of patent specification: 26.01.94 (51) Int. Cl.⁵: B23B 5/18

(21) Application number: 88904887.2

(22) Date of filing: 13.05.88

(86) International application number:
PCT/US88/01590

(87) International publication number:
WO 88/08765 (17.11.88 88/25)

(54) MONOTONIC CUTTING MACHINE.

(30) Priority: 13.05.87 US 49607

(43) Date of publication of application:
03.05.89 Bulletin 89/18

(45) Publication of the grant of the patent:
26.01.94 Bulletin 94/04

(84) Designated Contracting States:
AT BE CH DE FR GB IT LI LU NL SE

(66) References cited:
EP-A- 0 264 474
US-A- 2 287 449
US-A- 2 468 745
US-A- 3 271 910

(73) Proprietor: HEFFRON, Allan J.
2603 Blackhurst
Midland, MI 48640(US)

(72) Inventor: HEFFRON, Allan J.
2603 Blackhurst
Midland, MI 48640(US)

(74) Representative: Prato, Roberto et al
STUDIO TORTA Società Semplice
Via Viotti 9
I-10121 Torino (IT)

Note: Within nine months from the publication of the mention of the grant of the European patent, any person may give notice to the European Patent Office of opposition to the European patent granted. Notice of opposition shall be filed in a written reasoned statement. It shall not be deemed to have been filed until the opposition fee has been paid (Art. 99(1) European patent convention).

Description

The present invention relates to devices for machining parts, and more particularly to an apparatus for machining main bearings, crankpins, flanges, cheeks and the like of the crankshaft of an internal combustion engine, or machining any part having a round outside diameter on face.

Description of the Prior Art

The crankshaft is a well known element of a reciprocating piston engine or compressor. The crankshaft includes a plurality of main bearings on whose centers the crankshaft rotates, and a plurality of crankpins offset from the axis of the crankshaft. The pistons of the engine are connected to the crankpins by a plurality of connecting rods. The crankshaft thus changes the linear motion of the pistons to rotational motion, which is then transmitted to a drive means, transmission or the like.

Due to the high rotational speed and stresses encountered during operation, the crankshaft is forged metal (typically steel or ductile iron) for strength, and it is critical that the crankshaft be machined with great precision. The elements to be machined include the counterweight faces (cheeks) and diameters; the main bearing and crankpin bearing faces (sidewalls and thrust walls), diameters and undercuts; and other faces and diameters such as gear fits, oil seals, pulley fits, slingers, flanges, gears and reluctors.

Several methods are known for machining crankshafts from rough forgings or castings. However, each of the known methods has a number of associated drawbacks. Machining of the crankshafts by turning allows little flexibility for tooling changeover, requires frequent tool change and other maintenance, has poor swarf control and entails a high sound level in the shop. In contrast, machining by skiving (and/or turn broaching) has a high initial and rebuilding cost, requires a good deal of floor space, has no steady rest capability, and has little flexibility in machine configuration. Milling has also been employed for machining crankshafts, but does not readily allow the combining of several operations on the workpiece at the same location, and at the same time entails relatively expensive maintenance, is not useful for a number of crankshaft operations, requires special premachining of the crankshaft, and also has a relatively high initial and rebuilding cost. A last known method, grinding, is not often used to machine crankshafts because of high initial and rebuilding costs, low production capability, a high cost in perishable tools, significant tool change downtime, an inability to combine several operations, and an inability to achieve optimal cutting speeds, feed rates and tool dimen-

sional tolerances. The relatively high cost of machining crankshafts is a significant factor in the cost of manufacturing an engine, and therefore in manufacturing a motor vehicle. Indeed, one major cost and inefficiency in prior methods of manufacture has been the need to withdraw or reciprocate the machining edge, tool or implement from the area of the workpiece, before positioning and working a new workpiece.

The aforementioned advantages have been partially solved by an apparatus as disclosed in EP-A-0264474, the content thereof is to be considered as comprised in the state of the art pursuant to Article 54(3) and (4) EPC. EP-A-0264474 discloses an apparatus for machining a workpiece having a centerline, comprising a lathe bed, means for detachably mounting the workpiece to the lathe bed with the center line disposed in a predetermined direction, means for rotating the workpiece about said center line in a first rotational direction, an arbor having a longitudinal axis disposed parallel to the workpiece axis of rotation, means for rotating the arbor about its longitudinal axis in a second direction the same as the first direction, a planar disk perpendicularly mounted to the arbor and rotatable therewith so as to define a disk center and a sequential plurality of tools disposed on the circumference of the disk at distances from the disk center generally increasing in a stepwise, continuous fashion along a first portion of the disk circumference; the disk includes a cutout second portion on its circumference dimensioned so as to space for mounting and demounting the workpiece to and from the lathe bed. The scope of the present invention is to provide an apparatus which overcomes the aforementioned and other drawbacks in the prior devices, and also has improved operation with respect to the one disclosed in EP-A-0264474.

Summary of the present invention

Such a scope is attained by the present invention by providing an apparatus according to claim 1. Such an apparatus for machining a workpiece or part having a round outside diameter or face, such as a crankshaft, which provides superior combination of low initial and rebuilding cost, and high flexibility in tool changeover and alteration of configuration, while minimizing floor space and noise level. The apparatus of the present invention also combines several machine operations into a single step, and permits the use of optimum cutting speeds, cutting tool feed rates, cutter composition and cutting tool dimensional tolerances.

The apparatus of the present invention comprises a lathe bed and means rotatably mounting the workpiece, preferably a forged, precentered crankshaft, to the bed. The apparatus also includes

an arbor disposed parallel to the axis of rotation of the workpiece, rotatable in the same direction as the direction of rotation of the workpiece. At least one substantially planar disk is fixedly mounted on the arbor and rotatable in unison with it. The disk has a plurality of cutting tools disposed about part of its circumference, the tools being located at a generally increasing distance from the center of the disk in a stepwise fashion. The circumference of the disk also includes a second cutout portion between the radially outermost and innermost tools.

In operation, the disk is rotated to bring first the radially innermost tools and then the sequentially more outer tools, into contact with the same location on the rotating workpiece, in order to machine it. The workpiece and disk are each rotated at programmed, variable rates, in order to optimally match the cutting tool feed rate and the workpiece surface linear speed for each of the tools or groups of tools, for example, rough cutting, finish cutting, or dimensioning tools, depending upon the operations to be performed.

Once the radially outermost tool has cut the workpiece, the cutout portion of the disk circumference provides sufficient space for the ready removal of the workpiece from the lathe bed without the need to shift the disk and its associated cutting tools out of the way, or recycle it back to an initial, disengaged position. The workpiece is simply rotated to its initial position in a direct fashion. Thus, in addition to the previously mentioned advantages, the present invention avoids the tool slides or reciprocating carriers, and their associated wear, poor dimensional tolerances, and extra mechanisms, encountered in prior art broaching tools and the like.

Brief Description of the Drawing

A better understanding of the present invention will now be had upon reference to the following detailed description, when read in conjunction with the accompanying drawing, wherein like reference characters refer to like parts throughout the several views, and in which:

FIG. 1 is a front elevational view of the preferred embodiment of the present invention for a typical main bearing application;

FIG. 2 is a cross-sectional view taken along line 2-2 of FIG. 1;

FIG. 3 is a cross-sectional view taken along line 3-3 of FIG. 2;

FIG. 4 is a cross-sectional view taken along line 4-4 of FIG. 2;

FIG. 5 is a partial top view of the preferred embodiment of the present invention, similar to FIG. 1 except depicting a double end drive machine operating on the counterweight edges;

and

FIG. 6 is a partial side view of another preferred embodiment of the present invention depicting a double end drive machine operating on nos. 2 and 3 crankpins.

Detailed Description of the Preferred Embodiment of the Present Invention

With reference now to FIG. 1, the machining apparatus 10 of the present invention is thereshown first comprising a unitary outer casing 12. A lip 13 located about the perimeter of the casing 12 directs stray machining coolant or lubricant back to a swarf removal opening (not shown), discharged to a conventional chip disposal apparatus. The casing 12 includes a pair of spindle housings 14 made of reinforced, heavy walled, fine grain cast iron, mounted to the casing with cross keys to maintain alignment therewith. A pair of arbor housings 16 are formed integrally with the spindle housings 14, and are similarly made of fine grained cast iron.

A lathe bed for turning a workpiece to be machined is formed from a spaced pair of bearings 18 contained in the spindle housings 14 (only one bearing 18 being shown) and a pair of spindle quills 20 retractably mounted in each of the bearings 18. The quills each bear a spindle 22 thereon, centerable upon a workpiece 26 gripped and contained between the spindles 22 by a chuck 24 on the end of each of the spindles 22. The chucks 24 are hydraulically actuated, as is retraction of the quills 20, so that the workpiece 26 is readily mounted to and detached from the lathe bed. The chucks 24 each include a plurality of jaws 25 for gripping and dogging the workpiece 26. A tool arbor 28 is mounted to the arbor housings 16 by a pair of antifriction bearings 30 contained in the arbor housings 16.

The apparatus 10 of the present invention can preferably include a pair of steadyrests 27 engageable with a pair of journal surfaces 29 previously machined on the workpiece 26. Additionally, or alternatively, although not shown the apparatus 10 can include one or more conventional devices for manipulating the workpiece 26, such as a loader, transfer, bridge or other mechanism for moving the workpiece 26 with respect to the lathe bed.

Each of the spindles 22 and the arbor 28 rotate in the same direction, that is, each in a clockwise or counterclockwise direction, when viewed from the same end. Each spindle 22 is driven by a parallel set of precision helical gears (not shown) by a controlled frequency AC motor or other conventional motor through a sheave, V-belt and air brake arrangement. The workpiece 26 is thus driven on both of its ends. The arbor 28 is driven

by another controlled frequency AC motor or other conventional motor through a heavy duty compound worm gear and snubber arrangement (not shown), the latter for the purpose of preventing over rotation of the arbor 28. The drive connection between the means for driving the spindles 22 and the arbor 28 is characterized in being adapted to drive the spindles 22 and arbor 28 at different, variable rotation rates.

The arbor 28 preferably comprises a 250 millimeter diameter steel tube having a wall thickness of 50 millimeters. The outside diameter of the arbor 28 is ground and serves as the base for one or more rotatable cutter bodies attached to it. The cutter bodies preferably comprise a plurality of disks, such as the disks 32, 34 and 36 shown in FIG. 1. Rotation of the arbor 28 rotates the cutter bodies with respect to the workpiece 26, thereby performing machining operations upon it. The workpiece 26 is rotated at a speed greater than the rotational speed of the arbor 28, while one rotation of the arbor 28 is all that is necessary to complete the machining of the workpiece 26 performed by the apparatus 10.

With reference now to FIGS. 1 and 2, the structure of the cutting bodies is more particularly exemplified by the disk 32. A plurality of cutting tools 38 are disposed about the circumference 40 of the disk 32, increasing in radial distance from the center of the disk 32 from a radially innermost cutting tool 42 to a radially outermost cutting tool 44. The circumference 40 of the disk 32 also includes a cutout portion 46 bearing no cutting tools thereon, but instead providing a clearance space for the free passage of the workpiece 26 with respect to the disk 32. The disk 32 is formed from a plurality of disk segments 50 rigidly but detachably mounted to a disk hub 48. The position of the segments 50 with respect to the hub 48 are precisely maintained by engagement of a hub key 49 with the hub 48 and each of the segments 50. The hub 48 is similarly rigidly but detachably affixed to the arbor 28, and maintained in a precise position by an arbor key 51 engaging the arbor 28 and the hub 48.

In operation, the workpiece 26 rotates in, for example, a counterclockwise direction as indicated by the arrow 54. The disk 32 rotates in the same direction, for example, the counterclockwise direction, as indicated by the arrow 52. Rotation of the disk 32 thus brings first the radially innermost cutting tool 42 into engagement with the workpiece 26, and the other tools 38 sequentially into engagement with the workpiece 26, until the workpiece is finally engaged and machined by the radially outermost tool 44.

With reference now to FIG. 3, one method of attaching the segments 50 to the hub 48 is more

particularly shown. Each disk segment 50 includes an annular recess formed on the radially inner surface of the segment 50, which receives therein an annular projection 58 formed on the radially outer surface of the hub 48. Additionally, each of the segments 50 additionally includes at least a pair of apertures 60 formed therethrough in registry with a matching pair of threaded bores 62 formed in the hub 48, one either side of the projection 58. A screw 64 is disposed through each aperture 60 and is engaged in each of the bores 62, so as to rigidly yet detachably secure each segment 50 to the hub 48.

With reference now to FIGS. 3 and 4, two of the diverse plurality of tool bits useful in the present invention are shown. Preferably, the tools 38 are arranged around the circumference 40 of the disk 32 such that utility grade tool bits, such as a pair of bits 66 on the tool 42, are employed on the more radially inner tools 38, while precision grade tool bits such as a bit 68 on the tool 44 are employed on the more radially outer tools 38. This permits utility grade tool bits to be employed for roughing cuts and the like, that is, the less precise machining steps, while precision grade tool bits are employed for finished diameter cuts, burnishing and the like. The utility grade bits 66 and the precision grade bits 68 are attached to the respective disk segments 50 in any conventional fashion, for example, by screws or clamps (not shown).

The apparatus 10 of the present invention is particularly useful for the machining of the crankshaft of an internal combustion engine. Such a crankshaft is generally shown in FIG. 1, but its elements are of the arrangement more particularly shown in FIGS. 5 or 6. By way of example, a crankshaft 70 for a four cylinder automobile engine includes a coaxial plurality of main bearings 72 separating two pairs 74 and 76 of coaxial crankpins. The crankpins 74 and 76 are connected to the main bearings 72 by a plurality of arms 78. The arms 78 also connect a plurality of counterweights 80 to the main bearings 72, opposite the crankpins 74 and 76. During the machining operation, the diameters of the main bearings 72 and crankpins 74 and 76 must be machined to the appropriate diameters, rough ground and/or finish ground. Additionally, the faces of the counterweights 80 as well as the faces of the flanges must be sized to ensure that they do not interfere with rotation of the cranks upon the crankpins 74 and 76 during operation of the engine. Further, it is desirable to form a pair of webs 84 on each of the crankpins 76, at least on their radially outermost surface. Undercutting of the various members of the crankshaft 70 is also desirable.

The crankshaft 70 includes a first end 86 having a post formed thereon, and a flange or second

end 88 having a flange formed thereon. It is preferred that each end 86 and 88 of the crankshaft 70 has been previously centered, that is, a conical recess has been sunk into the end for establishing a centerline for rotation, machining and repair, and additionally includes a pair of reference diameters 90 and 92 machined thereon, respectively. While the reference diameters 90 and 92 can be machined according to the present invention, it is preferred that such diameters be provided on the crankshaft 70 prior to machining by the apparatus 10 of the present invention. For machining performed while the crankshaft 70 is rotated about the axis of the main bearings 72, the chucks 24 grip the diameters 90 and 92, by abutment of the jaws 25 thereagainst.

With reference again to FIG. 1, operation of the apparatus 10 of the present invention for machining the crankshaft 70 can now be readily understood. As disclosed, the apparatus 10 preferably incorporates three cutting disks 32, 34 and 36, corresponding one each to two of the main bearings 72 and the oil seal diameter 73 of the crankshaft 70. Initially, the quills 20 are retracted into the spindle housings 14 so as to allow the steadyrest 27 to support and position the crankshaft 70, the particular workpiece 26 being machined, between the spindles 22. The quills 20 are then actuated to bring the spindles 22 into abutment with the crankshaft 70, and the chucks 24 actuated to clamp the diameters 90 and 92 on the post end 86 and flange end 88 of the crankshaft 70. In this rest position, the cutout portions 46 of the circumferly 40 of the disk 32, as well as of the disks 34 and 36, are disposed adjacent the crankshaft 70. The crankshaft 70 is then rotated in the direction of the arrow 54 by the spindles 22 and the quills 20. Typically, while varied during operation, the rotational speed of the crankshaft will be about 1000 rpm. Once the crankshaft 70 is up to speed, the arbor 28 is rotated in order to bring the cutting tools 38 on the circumferly of each of the disks 32, 34 and 36 into engagement with the surface of the crankshaft 70, specifically, at each of the two main bearings 72 and oil seal diameter 73. The tools 38 on each of the disks 32, 34 and 36 are staggered so that only the tools 38 on one of the disks 32, 34 and 36 engage the crankshaft 70 at any given moment.

The arbor 28 is rotated (again, preferably at a variable speed) so that first the radially innermost tool 42 on each of the disks 32, 34 and 36 engages the spinning crankshaft 70, followed by engagement of the radially more outward tools 38 on each of the disks 32, 34 and 36, until the workpiece is finally engaged by the radially outermost tool 44 on each of the disks 32, 34 and 36. Additionally, in their travel toward the main bearings 72, the tools 38 will also machine (cheek) the faces of the adja-

cent arms 78 and counterweights 80.

When the arbor has rotated to a position where the cutout portion 46 of each of the disks 32, 34 and 36 are adjacent the crankshaft 70, rotation of the arbor 28 and crankshaft 70 are ceased. The chucks 24 are then actuated to disengage from the crankshaft 70, and the quills 20 retracted so as to withdraw the spindles 22 and the chucks 24 from the crankshaft 70. The support 27 is then actuated to remove the crankshaft 70 from the lathe bed, and replace it with a new workpiece to be machined. The arbor 28 is not rotated backwards or transversely withdrawn after cutting, but is instead maintained in position, the cutout portion 46 of each disk 32, 34 and 36 precluding interference with the loading and unloading of the crankshaft 70. The disks 32, 34 and 36 are thus in position to machine a new crankshaft 70 without being rotated in the reverse direction.

It is an advantage of the present invention, and it is preferred, that several different machining operations are performed during a single rotation of the arbor 28. For example, with respect to the main bearings 72, the machining operations performed by the sequential plurality of tools 38 can include rough cutting, rough undercutting, rough diameter turning, semi-finish undercutting, further rough diameter turning, finish undercutting and finish diameter cutting, as well as the cheeking mentioned earlier.

The apparatus 10 of the present invention can be used to perform various other machining operations. For example, with particular reference to FIG. 5, the preferred embodiment shown therein can be used to machine the radially outermost edges 98 of the counterweights 80. The crankshaft 70 is chucked on its end diameter 90 and 92 and rotated so as to present the edges 98 of the counterweights 80 for machining by two staggered pairs of rotatable cutting disks 100 and 102. For clarity, the disks 100 and 102 are shown in FIG. 5 as if they are mounted to different arbors, to show their engagement with all four counterweight edges 98. If the disks 100 and 102 are all mounted to the same arbor, then they would of course be located on the same side of the crankshaft 70; the disks 100, for example, would then be in a position diametrically opposed across the crankshaft 70 as compared to their position shown in FIG. 5.

With reference now to FIG. 6, another preferred embodiment of the present invention is thereshown which is useful for machining the crankpins 74 or 76 themselves, as well as the adjacent faces of the counterweights 80 and flanges 78. The chucks 24 of the apparatus 10 are replaced by a pair of pot chucks 94 and 96 centered on the spindles 22. The pot chucks 94 and 96 each include a radially offset partial slot 104 or 106, respectively, the slots 104

and 106 being disposed parallel to one another. The slots 104 and 106 have rounded bottoms (not shown) and are dimensioned to slidably receive coaxial diameters or bearings on the crankshaft 70, such as the nos. 2 and 4 main bearings 108 and 110, respectively. A pair of stops 112 and 114 on the pot chucks 94 and 96 abut the outer surface of the crankshaft 70 and prevent rotation of the crankshaft 70 relative to the chucks 94 and 96, that is, rotation in the bottom of the slots 104 and 106, when the crankshaft 70 is rotated during machining. The slots 104 and 106 are dimensioned, either by being offset a sufficient distance from the axis of rotation 116 of the spindles 22, or by possessing a length and angle with respect to the centers of the chucks 94 and 96, so that one set of crankpins 74 or 76 (the latter, in FIG. 6) is coaxially disposed on the spindle axis 116. The crankpins 76 are then rotated and machined in the same manner as the main bearings 72, by a pair of cutter disks 118 and 120. The other pair of crankpins 74 can be machined in a similar fashion, gripped in pot chucks (not shown) which are mirror-image reversals of the pot chucks 94 and 96. Of course, in practice, the apparatus 10 of this preferred embodiment of the invention crankshafts into and out of engagement with the slots 104 and 106 on the pot chucks 94 and 96.

Several features characterize each of these embodiments of the present invention. The rotational rates of each of the crankshaft 70 (or other workpiece 26) and the arbor 28 are varied with respect to one another so as to optimize the match of the feed rate of the tools 38 and the linear surface speed of the portion of the crankshaft 70 being machined, in dependence upon the type of cutting or machining performed by each individual tool 38. In the machining operations described above, upwards of a dozen different types of cutting tools can be employed on each disks, to provide as complete a treatment as possible of the machined portions of the crankshaft 70. It is also preferred in the present invention that, insofar as practicable, the tools 38 include indexable carbide tool bits of standard, interchangeable configuration. For example, it is preferred that sidewall and diameter finishing tools are mounted in adjustable preset cartridges. In this manner, tool inserts or bits can be readily changed while the arbor and cutters remain in tact on the machine, in contrast to the extensive downtime necessary in the prior art for replacement of the machine tooling.

It has also been found advantageous to employ a coolant or lubricating material in conjunction with the apparatus 10 of the present invention, although generally more for the purpose of swarf control than for cooling or lubrication. It is preferred that the apparatus include means for controlling the

temperature of the cutting disks and associated machinery, rather than employing a coolant, or mechanical temperature compensation, for control of the dimensional tolerances of the assembled machine.

The apparatus of the present invention possesses numerous advantages over the machining devices of the prior art. One significant advantage is a reduction in ambient noise, as a result of the absence of the heavy intermittent plunge cut encountered during use of conventional broaching, milling or turning machines. The apparatus of the present invention combines several machining operations into a single operation, reducing the time, energy and equipment necessary to complete the machining operation. The absence of any slide from a conventional broaching tool permits higher dimensional tolerances to be achieved, as compared to conventional machining devices. Most particularly, the apparatus permits a careful matching of the feed rate of the individual tools (by variably controlling the rotation rate of the arbor) with the linear surface speed of the workpiece being machined (also variable, and directly independent upon the rotational speed of the workpiece). Many other advantages exist as well.

Claims

Claims for the following Contracting States :
BE, CH, LU, NL, SE

1. An apparatus (10) for machining a workpiece (26), said workpiece having a centerline, said apparatus (10) comprising:
 - a lathe bed;
 - means (22,24) for detachably mounting said workpiece (26) to said lathe bed with said workpiece centerline disposed in a predetermined direction;
 - means for rotating said workpiece (26) on said centerline in a first rotational direction and at a variable rotational speed, thereby defining a workpiece axis of rotation;
 - an arbor (28) having a longitudinal axis disposed parallel to said workpiece axis of rotation;
 - means for rotating said arbor (28) about said longitudinal axis in only a second rotational direction the same as said first rotational direction and at a variable rate of speed;
 - a planar disk (32) perpendicularly mounted to said arbor (28) and being rotatable therewith so as to define a disk center; and
 - a sequential plurality of tools (38) disposed on the circumference of said disk (32) at distances from said disk center generally increasing in a stepwise, continuous fashion along a first portion of said disk circumference;

wherein said disk (32) includes a cutout second portion (46) on the circumfery of said disk dimensioned so as to space for mounting and demounting said workpiece (26) to and from said lathe bed.

2. An apparatus according to claim 1, wherein said means for detachably mounting said workpiece to said bed comprises a chuck (24) attached to said bed, receiving a part of said workpiece (26) therein. 10
3. An apparatus according to claim 1 or 2, characterized by comprising means coordinating said variable workpiece rotating means and said variable arbor rotating means so as to match the feed rate of said tools (38) on said disk (32) and the linear surface velocity of said workpiece (26) where engaged by said tools. 15
4. An apparatus according to claim 3, wherein during rotation of said disk, the rate of rotation of at least one of said workpiece (26) and said disk (32) is set by said coordinating means to at least two different values. 20
5. An apparatus according to any of the preceding claims, characterized by comprising a unitary outer casing (12) and wherein said means for rotating said workpiece (26) comprises a pair of spindle housings (14) mounted to said casing (12), a pair of work spindles (22) engageable with said workpiece (26), and a pair of retractable quills (20) contained one each in said spindle housings (14), to which said work spindles (22) are connected. 30
6. An apparatus according to claim 5, wherein said quills (20) are mounted to said spindle housings (14) by antifriction bearings (18). 35
7. An apparatus according to any of the preceding claims, characterized by comprising at least one additional disk (34,36) having, an associated plurality of tools disposed on its circumfery, and means for rotating said additional disk (34,36) in the same direction as said first-mentioned disk (32). 40
8. An apparatus according to claim 7, wherein said at least one additional disk is mounted to said arbor (28). 45
9. An apparatus in accordance with claim 7 or 8, characterized in that said additional disk (34,36) has a plurality of tools staggered in relation to the tools (38) on the first mentioned disk (32) such that at any given moment said 55

workpiece is engaged by a tool (38) on only one of said disks (32,34,36).

**Claims for the following Contracting States :
AT, DE, FR, GB, IT**

1. An apparatus for machining a workpiece, said workpiece (26) having a centerline, said apparatus comprising: 5
a lathe bed;
means (22,24) for detachably mounting said workpiece (26) to said lathe bed with said workpiece centerline disposed in a predetermined direction;
first drive means for rotating said workpiece on said centerline in a first rotational direction, thereby defining a workpiece axis of rotation;
an arbor (28) having a longitudinal axis disposed parallel to said centerline;
second drive means for rotating said arbor (28) about said longitudinal axis in a second rotational direction which is the same as said first rotational direction;
a planar disk (32) perpendicularly mounted to said arbor and rotatable therewith so as to define a disk center;
a sequential plurality of different tools (38) disposed on the circumfery of said disk (32) at distances from said disk center generally increasing in a stepwise, continuous fashion along a first portion of said disk circumfery,
said disk including a cutout second portion (46) on the circumfery of said disk (32) dimensioned so as to space for mounting and demounting said workpiece to and from said lathe bed, 20
said first drive means including means for rotating said arbor (28) through at most a single revolution about said longitudinal axis and at a variable speed, said second drive means including means for rotating said workpiece (26) at a variable rate of speed during a single revolution of said arbor (28), said apparatus (10) including means coordinating said variable workpiece rotating means and said variable arbor rotating means so as to match the feed rate of said tools (38) on said disk (32) and the linear surface velocity of said workpiece (26) where engaged by said tools. 25
2. An apparatus in accordance with claim 1, characterized by including at least a second disk (34,36) having a plurality of tools staggered in relation to the tools (38) on the first mentioned disk (32) such that at any given moment said workpiece (26) is engaged by a tool (38) on only one of said disks (32,34,36). 30

3. An apparatus in accordance with claim 1 or 2, characterized by comprising a unitary outer casing (12); said first drive means comprising a pair of spindle housings (14) mounted to said casing (12), a pair of work spindles (22) engageable with said workpiece (26), and a pair of retractable quills (20) contained one each in said spindle housings (14), to which said work spindles (22) are connected.
4. An apparatus in accordance with claim 3, characterized in that said quills (20) are mounted to said spindle housing by antifriction bearings (18).

Patentansprüche

Patentansprüche für folgende Vertragsstaaten
: BE, CH, LU, NL, SE

1. Vorrichtung (10) zur Bearbeitung eines Werkstücks (26), das eine Mittelachse aufweist, wobei die Vorrichtung (10) aufweist:
 - ein Drehbankbett;
 - eine Einrichtung (22, 24) zur abnehmbaren Lagerung des Werkstücks (26) an dem Drehbankbett, wobei die Werkstück-Mittelachse in einer vorgegebenen Richtung angeordnet ist;
 - eine Einrichtung, um das Werkstück (26) um die Mittelachse in einer ersten Drehrichtung und mit veränderbarer Drehgeschwindigkeit zu drehen, wodurch eine Werkstück-Drehachse definiert wird;
 - eine Welle (28) mit einer Längsachse, die parallel zu der Werkstückdrehachse angeordnet ist;
 - eine Einrichtung, um die Welle (28) um die Längsachse in nur einer zweiten Drehrichtung zu drehen, die die gleiche ist wie die erste Drehrichtung, mit veränderbarem Geschwindigkeitsverhältnis;
 - eine ebene Scheibe (32), die rechtwinklig auf der Welle (28) gelagert und mit dieser drehbar ist, um einen Scheibenmittelpunkt zu definieren; und
 - mehrere aufeinanderfolgende gesonderte Werkzeuge (38), die auf dem Umfang der Scheibe (32) in Abständen von dem Scheibenmittelpunkt angeordnet sind, die allgemein schrittweise kontinuierlich längs einem ersten Abschnitt des Scheibenumfangs zunehmen,
 - wobei die Scheibe (32) einen ausgeschnittenen zweiten Abschnitt (46) am Umfang der Scheibe aufweist, der so dimensioniert ist, daß er Raum für das Einbringen und Herausnehmen des Werkstücks (26) in und aus dem Drehbankbett schafft.

2. Vorrichtung nach Anspruch 1, wobei die Einrichtung zur abnehmbaren Lagerung des Werkstücks am Bett ein Spannfutter (24) aufweist, das an dem Bett angebracht ist und einen Teil des Werkstücks (26) aufnimmt.
3. Vorrichtung nach Anspruch 1 oder 2, dadurch gekennzeichnet, daß sie eine Einrichtung aufweist, die die verstellbare Werkstück-Drehrichtung und die verstellbare Wellendreheinrichtung so koordiniert, daß das Geschwindigkeitsverhältnis der Werkzeuge (38) auf der Scheibe (32) und die lineare Oberflächengeschwindigkeit des Werkstücks (26), wo es in Eingriff mit den Werkzeugen steht, aneinander angepaßt werden.
4. Vorrichtung nach Anspruch 3, wobei während der Drehung der Scheibe die Drehgeschwindigkeit mindestens des Werkstücks (26) und/oder der Scheibe (32) durch die Koordinationseinrichtung auf mindestens zwei verschiedene Werte eingestellt wird.
5. Vorrichtung nach einem der vorangehenden Ansprüche, dadurch gekennzeichnet, daß sie ein einheitliches äußeres Gehäuse (12) aufweist, daß die Einrichtung zur Drehung des Werkstücks (26) zwei Spindelgehäuse (14) aufweist, die an dem Gehäuse (12) gelagert sind, zwei Arbeitsspindeln (22), die mit dem Werkstück (26) in Eingriff bringbar sind, und zwei zurückziehbare Hohlwellen (20), die jeweils in einem der Spindelgehäuse (14) enthalten sind und mit denen die Arbeitsspindeln (22) verbunden sind.
6. Vorrichtung nach Anspruch 5, wobei die Hohlwellen (20) in den Spindelgehäusen (14) in reibungsarmen Lagern (18) gelagert sind.
7. Vorrichtung nach einem der vorangehenden Ansprüche, dadurch gekennzeichnet, daß sie mindestens eine zusätzliche Scheibe (34, 36) aufweist, mit einer Vielzahl von zugeordneten Werkzeugen, die an ihrem Umfang angeordnet sind, und eine Einrichtung zur Drehung der zusätzlichen Scheibe (34, 36) in der gleichen Richtung wie die erstgenannte Scheibe (32).
8. Vorrichtung nach Anspruch 7, dadurch gekennzeichnet, daß mindestens eine zusätzliche Scheibe auf der Welle (28) gelagert ist.
9. Vorrichtung nach Anspruch 7 oder 8, dadurch gekennzeichnet, daß die zusätzliche Scheibe (34, 36) eine Vielzahl von Werkzeugen aufweist, die relativ zu den Werkzeugen (38) auf

der erstgenannten Scheibe (32) so versetzt sind, daß in jedem Zeitpunkt das Werkstück mit einem Werkzeug (38) auf nur einer der Scheiben (32, 34, 36) in Eingriff steht.

Patentansprüche für folgende Vertragsstaaten : AT, DE, FR, GB, IT

1. Vorrichtung zur Bearbeitung eines Werkstücks (26), das eine Mittelachse aufweist, wobei die Vorrichtung aufweist:

ein Drehbankbett;
eine Einrichtung (22, 24) zur abnehmbaren Lagerung des Werkstücks (16) an dem Drehbankbett, wobei die Werkstück-Mittelachse in einer vorgegebenen Richtung angeordnet ist;
eine erste Antriebseinrichtung, um das Werkstück um seine Mittelachse in einer ersten Drehrichtung zu drehen, wodurch eine Werkstück-Drehachse definiert wird;
eine Welle (28) mit einer Längsachse, die parallel zu der Mittelachse angeordnet ist;
eine zweite Antriebseinrichtung, um die Welle (28) um die Längsachse in einer zweiten Drehrichtung zu drehen, die die gleiche ist wie die erste Drehrichtung;
eine ebene Scheibe (32), die rechtwinklig auf der Welle gelagert und mit dieser drehbar ist, um einen Scheibenmittelpunkt zu definieren;

mehrere aufeinanderfolgende gesonderte Werkzeuge (38), die auf dem Umfang der Scheibe (32) in Abständen von dem Scheibenmittelpunkt angeordnet sind, die allgemein schrittweise kontinuierlich längs einem ersten Abschnitt des Scheibenumfangs zunehmen,

wobei die Scheibe einen ausgeschnittenen zweiten Abschnitt (46) am Umfang der Scheibe (32) aufweist, der so dimensioniert ist, daß er Raum für das Einbringen und Herausnehmen des Werkstücks in und aus dem Drehbankbett schafft,

wobei die erste Antriebseinrichtung eine Einrichtung aufweist, um die Welle (28) um höchstens eine einzige Umdrehung um die Längsachse und mit veränderbarer Geschwindigkeit zu drehen, wobei die zweite Antriebseinrichtung eine Einrichtung aufweist, um das Werkstück (26) mit veränderbarer Geschwindigkeit während einer einzelnen Umdrehung der Welle (28) zu drehen, wobei die Vorrichtung (10) Einrichtungen aufweist, um die veränderbare Werkstückdreheinrichtung und die veränderbare Wellendreheinrichtung so zu koordinieren, daß die Geschwindigkeit der Werkzeuge (38) auf der Scheibe (32) an die lineare Oberflächengeschwindigkeit des Werkstücks (26) angepaßt wird, wo es in Eingriff mit den

Werkzeugen steht.

2. Vorrichtung nach Anspruch 1, dadurch gekennzeichnet, daß sie mindestens eine zweite Scheibe (34, 36) aufweist, die mehrere Werkzeuge aufweist, die relativ zu den Werkzeugen (38) auf der erstgenannten Scheibe (32) so versetzt sind, daß in jedem Zeitpunkt das Werkstück (26) mit einem Werkzeug (38) auf nur einer der Scheiben (32, 34, 36) in Eingriff steht.
3. Vorrichtung nach Anspruch 1 oder 2, dadurch gekennzeichnet, daß sie ein einheitliches äußeres Gehäuse (12) aufweist, daß die erste Antriebseinrichtung zwei Spindelgehäuse (14) aufweist, die an dem Gehäuse (12) angebracht sind, zwei Arbeitsspindeln (22), die mit dem Werkstück (26) in Eingriff bringbar sind, und zwei zurückziehbare Hohlwellen (20), die jeweils in einem der Spindelgehäuse (14) enthalten sind und mit denen die Arbeitsspindeln (22) verbunden sind.
4. Vorrichtung nach Anspruch 3, dadurch gekennzeichnet, daß die Hohlwellen (20) im Spindelgehäuse in reibungsarmen Lagern (18) gelagert sind.

Revendications

Revendications pour les Etats contractants suivants : BE, CH, LU, NL, SE

1. Appareil (10) pour usiner une pièce (26), ladite pièce à usiner présentant un axe, ledit appareil (10) comprenant :
- un banc de tour;
 - des moyens (22, 24) pour monter de façon amovible ladite pièce à usiner (26) sur ledit banc de tour, ledit axe de pièce à usiner étant disposé dans une direction prédéterminée;
 - un moyen pour entraîner en rotation ladite pièce à usiner (26) sur ledit axe, dans un premier sens de rotation et à une vitesse de rotation variable, de manière à définir un axe de rotation de la pièce à usiner;
 - un arbre (28) présentant un axe longitudinal disposé parallèlement audit axe de rotation de pièce à usiner;
 - un moyen pour entraîner en rotation ledit arbre (28) autour dudit axe longitudinal, dans un seul sens de rotation, le même que ledit premier sens de rotation et à une vitesse variable;
 - un disque planaire (32) monté perpendiculairement sur ledit arbre (28) et pouvant tourner avec ce dernier, de manière à définir un centre de disque; et

une pluralité séquentielle d'outils (38) disposés sur la circonférence dudit disque (32), à des distances, par rapport audit centre de disque, qui augmentent globalement de manière incrémentée, continue, le long d'une première partie de ladite circonférence de disque;

dans lequel ledit disque (32) comprend une deuxième partie (46) découpée sur la circonférence dudit disque, dimensionnée de manière à laisser un espace pour permettre le montage et le démontage de ladite pièce à usiner (26) sur et depuis ledit banc de tour.

2. Appareil selon la revendication 1, dans lequel ledit moyen pour monter de façon amovible ladite pièce à usiner sur ledit banc comprend un mandrin (24) fixé audit banc, recevant une partie de ladite pièce à usiner (26) en son sein.
3. Appareil selon la revendication 1 ou 2, caractérisé par le fait de comprendre un moyen coordonnant ledit moyen de rotation de pièce à usiner variable et ledit moyen de rotation d'arbre variable, de manière à s'adapter à la cadence d'amenée desdits outils (38) sur ledit disque (32) et à la vitesse en surface linéaire de ladite pièce à usiner (26), lorsqu'elle est mise en prise avec lesdits outils.
4. Appareil selon la revendication 3, dans lequel, durant la rotation dudit disque, la vitesse de rotation d'au moins l'un des éléments entre ladite pièce à usiner (26) et ledit disque (32) est établie, par ledit moyen de coordination, à au moins deux valeurs différentes.
5. Appareil selon l'une quelconque des revendications précédentes, caractérisé par le fait de comprendre un carter extérieur monobloc (12) et dans lequel ledit moyen pour faire tourner ladite pièce à usiner (26) comprend un couple de boîtiers de tourillon (14) montés sur ledit carter (12), un couple de tourillons de travail (22) susceptibles de venir en contact avec ladite pièce à usiner (26) et un couple d'arbres creux rétractables (20) contenus chacun dans lesdits boîtiers de tourillon (14), auxquels sont reliés lesdits tourillons de travail (22).
6. Appareil selon la revendication 5, dans lequel lesdits arbres creux (20) sont montés sur lesdits boîtiers de tourillon (14) au moyen de paliers à roulements (18).
7. Appareil selon l'une quelconque des revendications précédentes, caractérisé par le fait de comprendre au moins un disque (34, 36) supplémentaire, présentant une pluralité associée

d'outils disposés sur sa circonférence et un moyen pour faire tourner ledit disque (34, 36) supplémentaire dans le même sens que ledit disque (32) précité.

8. Appareil selon la revendication 7, dans lequel ledit au moins un disque supplémentaire est monté sur ledit arbre (28).
9. Appareil selon la revendication 7 ou 8, caractérisé en ce que ledit disque (34, 36) supplémentaire présente une pluralité d'outils disposés en quinconce par rapport aux outils (38) situés sur le disque (32) précité, de manière qu'à un moment donné quelconque, ladite pièce à usiner soit mise en prise avec un outil (38), sur un seul desdits disques (32, 34, 36).

Revendications pour les Etats contractants suivants : AT, DE, FR, GB, IT

1. Appareil pour usiner une pièce, ladite pièce à usiner (26) présentant un axe, ledit appareil comprenant :
 - un banc de tour;
 - des moyens (22, 24) pour monter de façon amovible ladite pièce à usiner (26) sur ledit banc de tour, ledit axe de pièce à usiner étant disposé dans une direction prédéterminée;
 - un premier moyen d'entraînement en rotation de ladite pièce à usiner sur ledit axe, dans un premier sens de rotation, de manière à définir un axe de rotation de la pièce à usiner;
 - un arbre (28) présentant un axe longitudinal disposé parallèlement audit axe;
 - un deuxième moyen d'entraînement en rotation dudit arbre (28) autour dudit axe longitudinal, dans un deuxième sens de rotation, qui est le même que ledit premier sens de rotation;
 - un disque planaire (32) monté perpendiculairement sur ledit arbre et pouvant tourner avec ce dernier, de manière à définir un centre de disque; et
 - une pluralité séquentielle d'outils (38) différents, disposés sur la circonférence dudit disque (32), à des distances, par rapport audit centre de disque, qui augmentent globalement de manière incrémentée, continue le long d'une première partie de ladite circonférence de disque,
 - ledit disque comprenant une deuxième partie (46) découpée sur la circonférence dudit disque (32), dimensionnée de manière à laisser un espace pour permettre le montage et le démontage de ladite pièce à usiner sur et depuis ledit banc de tour,
 - ledit premier moyen d'entraînement com-

prenant un moyen pour faire tourner ledit arbre (28) d'un seul tour au plus, autour dudit axe longitudinal et à une vitesse variable, ledit deuxième moyen d'entraînement comprenant un moyen pour faire tourner ladite pièce à usiner (26) à une vitesse variable durant un seul tour dudit arbre (28),

5

ledit appareil (10) comprenant un moyen coordonnant ledit moyen de rotation de pièce à usiner variable et ledit moyen de rotation d'arbre variable, de manière à s'adapter à la cadence d'amenée desdits outils (38) sur ledit disque (32) et à la vitesse en surface linéaire de ladite pièce à usiner (26), lorsqu'elle est en prise avec lesdits outils.

10

15

2. Appareil selon la revendication 1, caractérisé par le fait de comprendre au moins un deuxième disque (34, 36), présentant une pluralité associée d'outils disposés en quinconce par rapport aux outils (38) sur le disque (32) précité, de manière qu'à un moment donné quelconque, ladite pièce à usiner (26) soit mise en prise avec un outil (38), sur un seul desdits disques (32, 34, 36).

20

25

3. Appareil selon la revendication 1 ou 2, caractérisé par le fait de comprendre un carter extérieur monobloc (12); ledit premier moyen d'entraînement comprenant un couple de boîtiers de tourillon (14) montés sur ledit carter (12), un couple de tourillons de travail (22) susceptibles de venir en prise avec ladite pièce à usiner (26) et un couple d'arbres creux rétractables (20) contenus chacun dans lesdits boîtiers de tourillon (14), auxquels sont reliés lesdits tourillons de travail (22).

30

35

4. Appareil selon la revendication 3, caractérisé en ce que lesdits arbres creux (20) sont montés sur ledit boîtier de tourillon, par des paliers à roulements (18).

40

45

50

55

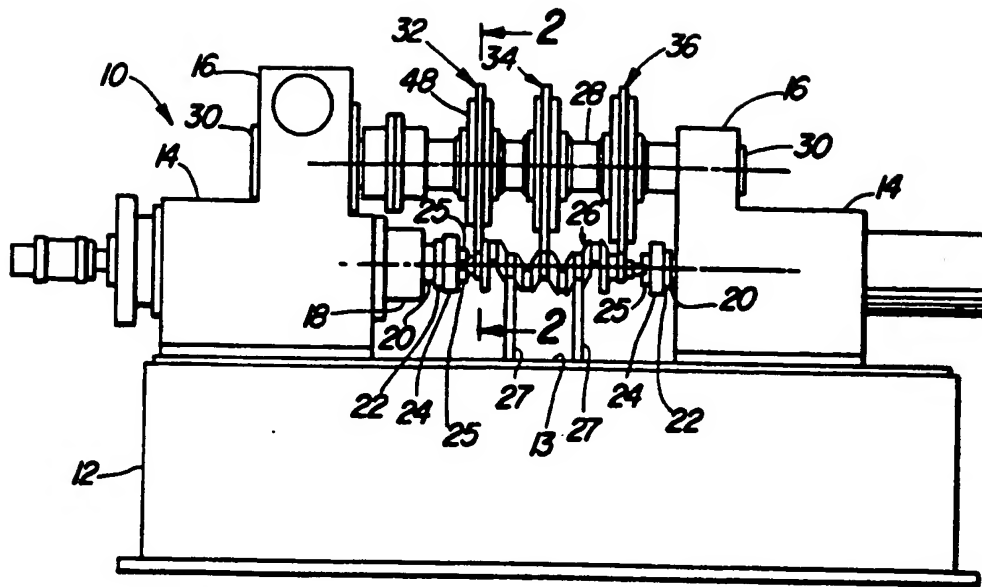


Fig-1

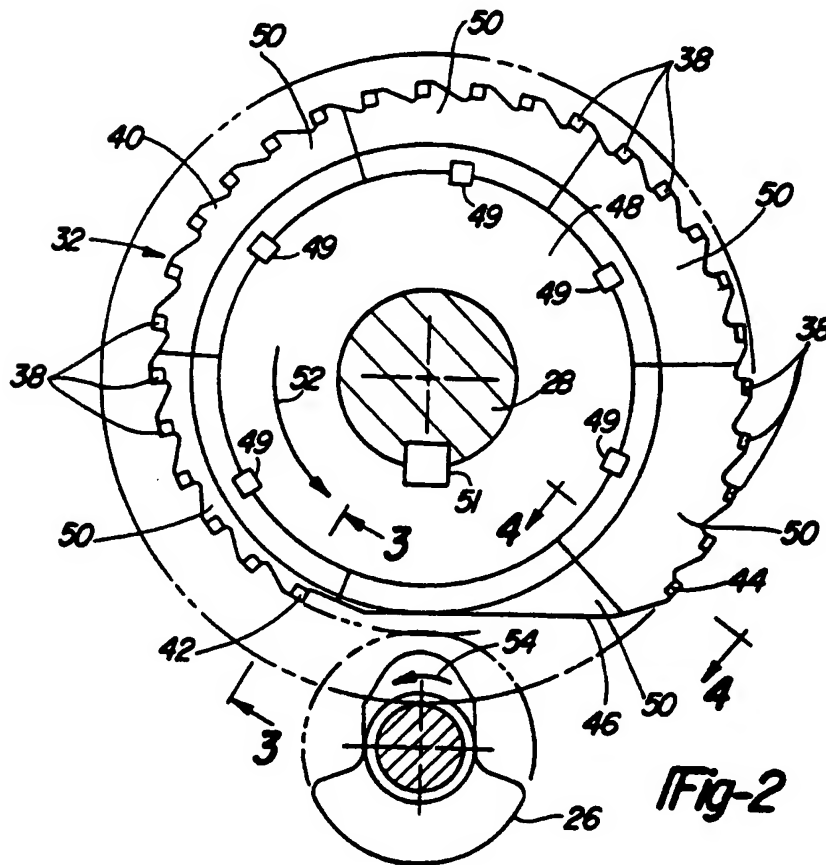


Fig-2

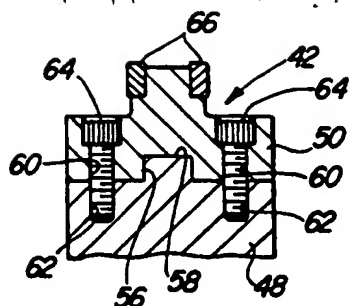


Fig-3

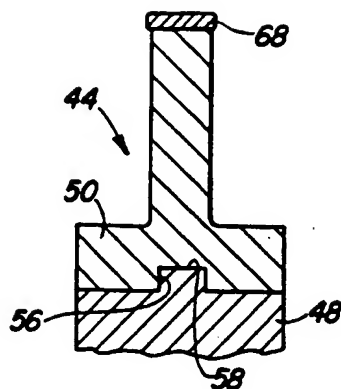


Fig-4

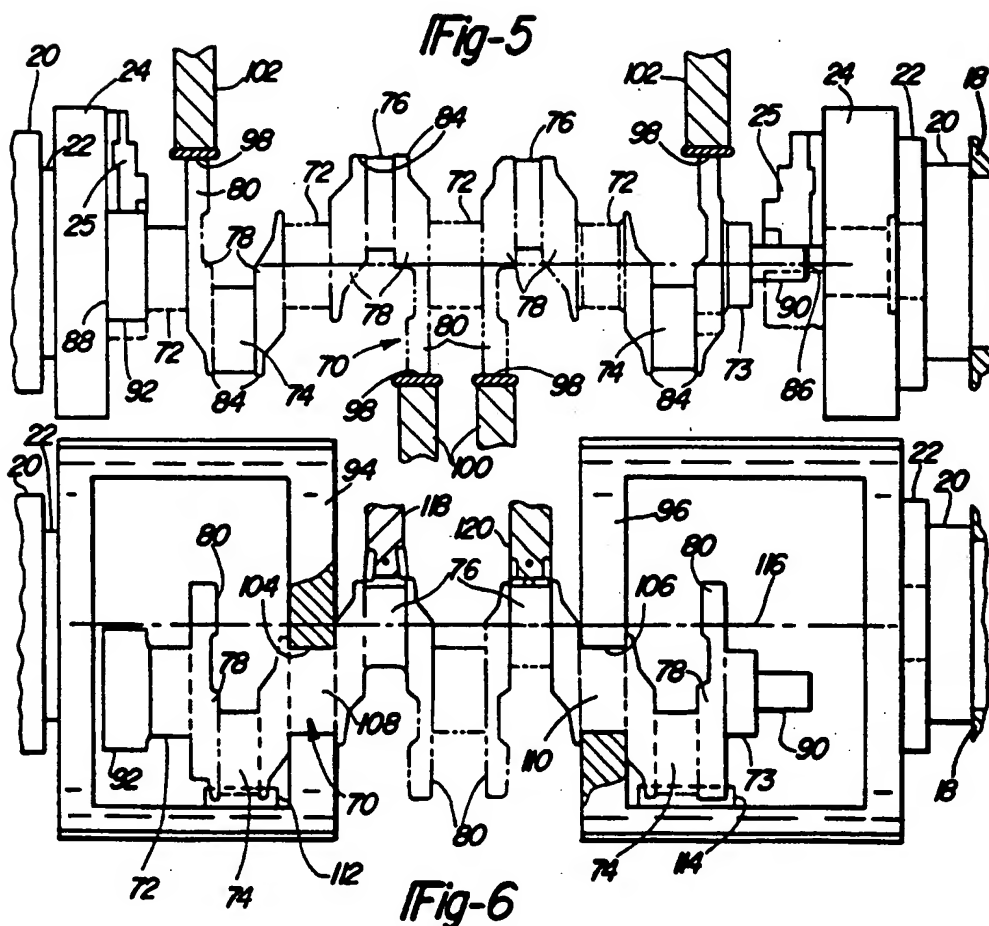


Fig-6

THIS PAGE BLANK (USPTO)

**This Page is Inserted by IFW Indexing and Scanning
Operations and is not part of the Official Record**

BEST AVAILABLE IMAGES

Defective images within this document are accurate representations of the original documents submitted by the applicant.

Defects in the images include but are not limited to the items checked:

- ☐ **BLACK BORDERS**
- ☐ **IMAGE CUT OFF AT TOP, BOTTOM OR SIDES**
- ☒ **FADED TEXT OR DRAWING**
- ☒ **BLURRED OR ILLEGIBLE TEXT OR DRAWING**
- ☐ **SKEWED/SLANTED IMAGES**
- ☐ **COLOR OR BLACK AND WHITE PHOTOGRAPHS**
- ☐ **GRAY SCALE DOCUMENTS**
- ☐ **LINES OR MARKS ON ORIGINAL DOCUMENT**
- ☐ **REFERENCE(S) OR EXHIBIT(S) SUBMITTED ARE POOR QUALITY**
- ☐ **OTHER:** _____

IMAGES ARE BEST AVAILABLE COPY.

As rescanning these documents will not correct the image problems checked, please do not report these problems to the IFW Image Problem Mailbox.

THIS PAGE BLANK (USPTO)